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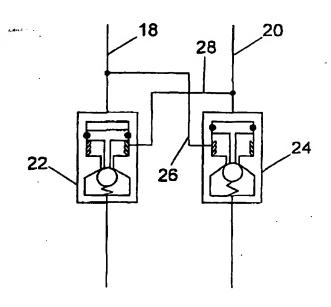
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(54) Title: MULTI-LINE BACK PRESSURE CONTROL SYSTEM

(57) Abstract

A multi-line back pressure control system for providing two way hydraulic line movement while maintaining back pressure control. Check valves are integrated in hydraulic fluid control lines extending downhole into a wellbore. Each check valve is pilot operated with pressure from another hydraulic line to selectively open the lines for two way fluid communication. Removal of the pilot pressure closes the check valves to provide passive back pressure control against catastrophic wellbore events. Pilot pressure operation between multiple pressurized lines can be provided with valves such as three-way, three-position piloted valves.



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MULTI-LINE BACK PRESSURE CONTROL SYSTEM

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BACKGROUND OF THE INVENTION

The present invention relates to a system for controlling downhole well tools to produce hydrocarbons from a wellbore. More particularly, the invention relates to a back pressure control system providing safe operation in multiple hydraulic control lines.

Downhole well tools control, select and regulate the production of hydrocarbon fluids and other fluids produced downhole from subterranean formations. Downhole well tools such as sliding sleeves, sliding side doors, interval control lines, safety valves, lubricator valves, chemical injection subs, and gas lift valves are representative examples of such tools. Well tools are typically controlled and powered from the wellbore surface by pressurizing hydraulic lines which extend from a Christmas Tree or other wellhead and into the wellbore lower end.

Dual pressure barriers in hydraulic lines are preferred to prevent hydraulic line failure during a wellbore catastrophic event. Dual pressure barrier systems have an active and a passive barrier. active barrier typically comprises a valve located at the Christmas Tree or wellhead, and the passive barrier

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typically comprises a check valve located in the 1 hydraulic line below the wellhead. 2 The check valve restricts fluid flow in one direction as the hydraulic 3 4 fluid, chemicals or other fluids are pumped downhole into the hydraulic line. 5 The fluids pressurize an actuator in a single operation or are discharged into 6 7 the tubing or wellbore annulus through an exit port or valve. 8 Certain tools such as safety valves require fluid 9 10 flow control in opposite directions. However, safety 11 valves do not internally provide dual barrier capabilities because such barriers would resist two-way 12 13 fluid flow. Because safety valves do not provide a 14 passive well control barrier, significant design effort has been made to enhance the reliability of safety 15 valve operation. 16 Safety valves have been designed with metal-to-metal fittings, metal dynamic seals, rod 17 piston actuators, and other features designed to 18 provide reliable operation during a catastrophic event 19 in the wellbore. Other safety valves use springs, 20 21 annulus fluid pressure, or tubing fluid pressure to 22 provide the restoring force necessary to return the 23 closure mechanism to the original position. 24 Downhole well tool actuators generally comprise 25 short term or long term devices. Short term devices include one shot tools and tools having limited 26 operating cycles. Hydraulically operated systems have 27 28 mechanical mechanisms with simple shear pins or complex 29 mechanisms performing over multiple cycles. Actuation signals are provided through mechanical, direct 30 31 pressure, pressure pulsing, electromagnetic, and other 32 mechanisms. The control mechanism may involve simple 33 mechanics, fluid logic controls, timers, or

springs, differential pressure, hydrostatic pressure,

or locally generated mechanisms. Long term devices

Motive force can be provided through

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electronics.

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provide virtually unlimited operating cycles and are designed for operation through the well producing life. One long term device provides a fail safe operating capabilities which closes with spring powered force

capabilities which closes with spring powered force when the hydraulic line pressure is lost. Combinati

6 electrical and hydraulic powered systems have been

7 developed for downhole use.

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Control for a downhole tool can be provided by 8 9 connecting a single hydraulic line to a tool such as an 10 internal control valve ("ICV") or a lubricator valve, 11 and by discharging hydraulic fluid from the line end 12 into the wellbore. This technique has several 13 limitations as the hydraulic fluid exits the wellbore 14 because of differential pressures between the hydraulic 15 line and the wellbore. The discharge of hydraulic fluid into the wellbore comprises an undesirable 16 environmental discharge, and the fluid discharge risks 17 18 backflow and particulate contamination in the hydraulic 19 Additionally, the setting depths are limited by the maximum pressure that a pressure relief valve 20 can hold between the differential pressure between the 21 22 control line pressure and the production tubing. of these limitations effectively restrict single line 23 hydraulic systems to relatively low differential 24 25 pressure applications such as lubricator valves and 26 sliding sleeves.

To overcome these limitations, a second hydraulic line can be installed to return hydraulic fluid to the wellbore surface through a closed loop. In United States Patent No. 4,942,926 to Lessi (1990), dual hydraulic lines provided tool operation in two directions. In United States Patent No. 3,906,726 to Jameson (1975), a manual control disable valve and a manual choke control valve controlled hydraulic fluid flow on either side of a piston head. In United States Patent No. 4,197,879 to Young (1980) and in 4,368,871

to Young (1983), two hydraulic lines controlled a

- 2 lubricator valve during well test operations. In all
- 3 of these tools, two hydraulic lines are inefficient
- 4 because the additional hydraulic lines increase sealing
- 5 problems and reduce the available space through packers
- 6 and wellheads. Additionally, passive barrier
- 7 protection for each hydraulic line is not possible
- 8 because of the return fluid flow from the well tool to
- 9 the surface.

Accordingly, a need exists for an improved system capable of providing back pressure control in systems having multiple hydraulic lines. The system should be reliable, adaptable to different tool configurations and combinations, and should provide passive back flow containment for downhole well tools.

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SUMMARY OF THE INVENTION

The present invention provides an apparatus for 18 19 providing back pressure control in at least two 20 hydraulic lines extending downhole in a wellbore. 21 apparatus comprises a check valve engaged with each of 22 the hydraulic lines in a closed initial position, wherein each of said check valves prevents pressurized 23 fluid downhole of the check valves from moving upstream 24 25 of the check valves, and hydraulic means operable with 26 the fluid pressure in a hydraulic line to selectively 27 open a check valve engaged with another of the 28 hydraulic lines to permit two-way fluid communication 29 through the check valve. The hydraulic means is 30 further operable when the hydraulic line fluid pressure 31 is reduced to return the check valve to the initial 32 position. 33

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valve or control valve combination having fewer valves than hydraulic lines.

In another embodiment of the invention, the 3 apparatus can selectively open fluid flow through 4 hydraulic lines extending between a wellbore surface 5 and a downhole tool. The apparatus can comprise a 6 check valve engaged with each hydraulic line in a 7 closed initial position where each of the check valves 8 prevents pressurized fluid downhole of the check valve 9 10 from moving upstream of said check valve, a hydraulic 11 means operable with the fluid pressure in a hydraulic line to selectively open a check valve engaged with 12. 13 another hydraulic line to permit two-way fluid 14 communication through the check valve, and a controller 15 engaged with the hydraulic lines for selectively 16 pressurizing at least one of the hydraulic lines to 17 operate said hydraulic means and to open a check valve engaged with another of the hydraulic lines. 18

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BRIEF DESCRIPTION OF THE DRAWINGS

21 Figure 1 illustrates engagement of a check valve 22 in a hydraulic line.

Figure 2 illustrates two hydraulic lines engaged having a pilot opening feature.

25 Figure 3 shows a three-way three-position valve.

Figure 4 illustrates a three hydraulic line application of the invention, wherein a valve is associate with each check valve.

Figure 5 illustrates a four hydraulic line application of the invention.

Figure 6 illustrates another application of the invention to a three hydraulic line system.

Figure 7 illustrates another application of the invention to a four hydraulic line system.

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| 1 | DESCRIPTION OF THE PREFERRED EMBODIMENTS |
|-----|---|
| 2 | The present invention provides passive back |
| 3 . | pressure control in multiple hydraulic lines, and is |
| 4 | adaptable to systems having two or more hydraulic |
| 5 | lines. The invention facilitates the creation of |
| 6 | hydraulic line systems providing control functions and |
| 7 | power requirements for the actuation of downhole well |
| 8 | tools. |
| 9 | Figure 1 illustrates the placement of conventional |
| 10 | back check valve 14 in hydraulic fluid line 16. |
| 11 | Hydraulic line 16 can extend from the wellbore surface |
| 12 | to engagement located downhole in the wellbore. As |
| 13 | illustrated, the direction of fluid flow can move in |
| 14 | one direction and is prevented from flowing in the |
| 15 | opposite direction. Figure 2 illustrates the |
| 16 | application of the invention to two hydraulic fluid |
| 17 | lines 18 and 20, wherein pilot operated check valves 22 |
| 18 | and 24 are integrated in fluid lines 18 and 20. Check |
| 19 | valves 22 and 24 operate as conventional check valves |
| 20 | to prevent fluid flow upwards from the lower end of |
| 21 | fluid lines 18 and 20. However, pilot operated check |
| 22 | valves 22 and 24 perform a different function when |
| 23 | combined with another fluid pressure source. When |
| 24 | fluid line 18 is pressurized, fluid moves downwardly |
| 25 | through check valve 22 and is further directed through |
| 26 | line 26 to check valve 24 to open check valve 24 to |
| 27 | two-way fluid flow. Similarly, the separate operation |
| 28 | of fluid line 20 moves fluid downwardly through check |
| 29 | valve 24 and is further directed through line 28 to |
| 30 | open check valve 22 to provide two-way fluid flow. |
| 31 | When the fluid pressure within line 18 is removed, the |
| 32 | pilot function for valve 24 is removed and valve 24 |
| 33 | closes to provide a passive pressure barrier. When the |
| 34 | fluid pressure within line 20 is removed, the pilot |
| 35 | function for valve 22 is removed and valve 22 closes to |
| 36 | provide a passive pressure barrier. |

| 1 | The extension of the invention to more than two |
|----|---|
| 2 | hydraulic lines is accomplished by incorporating a |
| 3 | valve for providing control over the pressure |
| 4 | communication or flow of fluid from multiple lines. |
| 5 | One such valve is illustrated in Figure 3, wherein |
| 6 | three-way, three-position piloted valve 29 has two |
| 7 | positions and three ports. Two ports comprise inlet |
| 8 | ports and the third comprises an outlet port. An |
| 9 | internal, free floating check ball senses flow and |
| 10 | pressure from the two inlet ports and closes the lessor |
| 11 | flow inlet port in favor of the greater flow inlet |
| 12 | port. In this manner, shuttle valve 29 automatically |
| 13 | provides a switching function between multiple lines |
| 14 | without requiring electrically operated solenoid |
| 15 | valves, additional hydraulic lines, electronic |
| 16 | controls, or other combinations conventionally used. |
| 17 | Different combinations of pilot activated check valves |
| 18 | and hydraulic switching valves such as shuttle valve 29 |
| 19 | can be connected in series or in parallel in various |
| 20 | configurations and combinations to accomplish different |
| 21 | operating functions. This combination provides unique |
| 22 | flexibility in providing back pressure control in |
| 23 | complex hydraulic operating systems. |
| 24 | Figure 4 illustrates a three hydraulic line system |
| 25 | wherein pilot check valves 30, 32 and 34 are integrated |
| 26 | with hydraulic lines 36, 38 and 40 to provide passive |
| 27 | back pressure control. Non-selective valves 42, 44 and |
| 28 | 46 are integrated into the system to selectively |
| 29 | provide the pilot function for check valves 30, 32 and |
| 30 | 34. Pressurization of line 36 opens check valve 30 and |
| 31 | further operates valve 44 to open check valve 32, and |
| 32 | operates valve 46 to open check valve 34. Release of |
| 33 | the pressure for line 36 causes check valves 30, 32 and |
| 34 | 34 to close lines 36, 38 and 40. Similarly, |
| 35 | pressurization of line 38 opens check valve 32, |
| 36 | operates valve 42 to open check valve 30, and further |

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1 operates valve 46 to open check valve 34. Release of 2 the pressure for line 38 causes check valves 30, 32 and 34 to close lines 36, 38 and 40. 3 Pressurization of line 40 accomplishes a similar function of opening 4 5 lines 36, 38 and 40. The dual pressurization of two lines such as lines 36 and 38 opens check valves 30 and 6 32 and operates valve 46 to open check valve 34 because 7 8 pressure from line 36 or line 38 will move through valve 46 to open check valve 34. 9

Figure 5 illustrates another embodiment of the invention applied to a four line system having lines 48, 50, 52 and 54, check valves 56, 58, 60 and 62, and valves 64, 66, 68, 70, 72, 74 and 76. Pressurization of line 48 opens check valve 56, operates valve 66 to operate valve 72 to open check valve 58, operates valve 68 to operate valve 74 to open check valve 60 and to operate valve 76 to open check valve 62. fashion, the pressurization of line 48 opens all four check valves 56, 58, 60 and 62. Similarly, the pressurization of line 52 opens check valve 60, operates valve 64 to operate valve 70 to open check valve 56, operates valve 66 to operate valve 72 to open check valve 58, and operates valve 76 to open check Withdrawal of pressure in line 52 causes each check valve to return to the initial closed position.

Figure 6 illustrates another combination of components for a three line isolation system to selectively open and close lines 36, 38 and 40 with check valves 30, 32 and 34. Valves 78 and 80 provide the functional operation provided by the three valves identified in Figure 4. Valves 78 and 80 provide a package for simultaneously opening check valves 30, 32 and 34. When line 36 or line 38 is pressurized, such hydraulic fluid line pressure operates valve 78 to operate valve 80 to open the check valves. When line

1 40 is pressurized, valve 80 is operated to open the check valves.

Figure 7 illustrates another embodiment of a four line isolation system to selectively open and close lines 48, 50, 52 and 54 with check valves 56, 58, 60 and 62. Valves 82, 84, and 86 provide the functional operation provided by the seven similar valves shown in Figure 5. When line 48 or line 50 is pressurized, such line pressure operates valve 82 to operate valve 84 and to operate valve 86 to open check valves 56, 58, 60 and When line 52 is pressurized, valve 84 operates valve 86 to open the check valves. When line 54 is pressurized, valve 86 is operated to open the check valves.

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The invention is particularly suited to systems requiring hydraulic fluid reliability to the control of downhole well tools by uniquely utilizing hydraulics with logic circuitry. Such logic circuitry is analogous to electrical and electronics systems, and can incorporate Boolean Logic using "AND" and "OR" gate combinations.

The invention is particularly suitable for use with digital-hydraulic control systems serving multiple well control devices. In such system, pressure is applied in a coded sequence to several hydraulic lines. The coded sequence automatically selects one of the well control devices and provides independent operation of the well control device. Instead of discharging hydraulic fluid into the tubing or wellbore, excess fluid is returned up one of the unpressurized hydraulic lines. To permit return flow of the excess fluid, a system must permit such return flow through one or more hydraulic lines, and this return flow is provided by controlling the opening of the pilot operated check valves.

The invention provides passive back check valves

If one or more of the lines

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on each hydraulic line.

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invention.

are pressurized from the wellbore surface, the back 2 check valves in the unpressurized lines are temporarily opened with pilot pistons activated by the pressurized 5 In this configuration, the passive barriers provided by the back check valves are temporarily opened for two-way fluid communication to permit single 7 tool operation or to permit selected tool operation for different combinations. After the pressure in a 9 hydraulic line is removed and the line pressure is bled 10 down or otherwise reduced, the back check valve on such 11 hydraulic line closes to prevent fluid flow in such 13 direction. Passive back pressure control is maintained 14 because pressure from below does not open the back check valve, and the piloting pressure to open the back 15 16 check valves is only provided by hydraulic line 17 pressure above the valve. Although the invention has been described in terms 18 19 of certain preferred embodiments, it will become 20 apparent to those of ordinary skill in the art that 21 modifications and improvements can be made to the 22 inventive concepts herein without departing from the 23 scope of the invention. The embodiments shown herein

are merely illustrative of the inventive concepts and

should not be interpreted as limiting the scope of the

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1 WHAT IS CLAIMED IS:

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1. An apparatus for providing back pressure control in at least two hydraulic lines extending downhole in a wellbore, comprising:

a check valve engaged with each of the hydraulic lines in a closed initial position, wherein each of said check valves prevents pressurized fluid downhole of said check valves from moving upstream of said check valves; and

hydraulic means operable with the fluid pressure in a hydraulic line to selectively open a check valve engaged with another of the hydraulic lines to permit two-way fluid communication through said check valve, wherein said hydraulic means is further operable when said hydraulic line fluid pressure is reduced to return said check valve to said initial position.

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2. An apparatus is recited in Claim 1, wherein each
 check valve comprises a pilot operated check valve.

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An apparatus as recited in either of Claims 1 or
 therein said hydraulic means comprises a pilot
 mechanism for each of said check valves.

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4. An apparatus as recited in any preceding Claim,
wherein increased fluid pressure in a hydraulic line
further opens the check valve engaged with such
hydraulic line to permit two-way communication through

30 said check valve.

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5. An apparatus as recited in any preceding Claim, further comprising at least three check valves each engaged with a separate hydraulic line, and wherein said hydraulic means comprises a control valve engaged

36 with two of said hydraulic lines for selectively

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1 communicating fluid pressure in one of two hydraulic

lines to open the check valve engaged with said third

3 hydraulic line.

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5 6. An apparatus as recited in Claim 5, wherein said

6 hydraulic means comprises a first control valve engaged

7 with the first and second hydraulic lines and with a

8 second control valve engaged with the third hydraulic

9 line, and wherein said second control valve is operable

in response to fluid pressure in the third hydraulic

line to open all three check valves, and wherein said

12 second control valve is further operable in response to

said first control valve to open all three check

14 valves.

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16 7. An apparatus as recited in any preceding Claim,

wherein said hydraulic means comprises two or more

18 three-way three-position valves each operable in

19 response to fluid pressure from one of two hydraulic

20 lines to engage and open one of said check valves for

21 permitting two-way fluid communication through said

22 check valve.

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24 8. An apparatus as recited in Claim 7, wherein each

three-way three-position valve is operable to open all

of said check valves for permitting two-way fluid

27 communication through said check valves.

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An apparatus as recited in any preceding Claim,

30 wherein said hydraulic means comprises at least three

control valves each engaged with at least one hydraulic

32 line and with at least one of said other control

33 valves, wherein each control valve is operable in

34 response to fluid pressure from one of said hydraulic

35 lines or other control valves to open at least one of

36 said check valves.

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1 10. An apparatus as recited in Claim 9, wherein one of 2 said control valves comprises a master control valve

- engaged with each hydraulic line and with each of said
- 4 check valves so that hydraulic fluid pressure in one of
- 5 the hydraulic lines is transmitted through said master
- 6 control valve to open all of said check valves for two-
- 7 way fluid communication.

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9 11. An apparatus for selectively opening fluid flow 10 through hydraulic lines extending between a wellbore 11 surface and a downhole tool, comprising:

a check valve engaged with each hydraulic line in a closed initial position, wherein each of said check valves prevents pressurized fluid downhole of said check valve from moving upstream of said check valve;

hydraulic means operable with the fluid pressure in a hydraulic line to selectively open a check valve engaged with another hydraulic line to permit two-way fluid communication through said check valve; and

a controller engaged with the hydraulic lines for selectively pressurizing at least one of the hydraulic lines to operate said hydraulic means to open a check valve engaged with another of the hydraulic lines.

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12. An apparatus as recited in Claim 11, wherein each check valve comprises a back flow device having an override.

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29 13. An apparatus as recited in either of Claims 11 or 30 12, wherein said hydraulic means comprises an override 31 engaged with each of said check valves.

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- 33 14. An apparatus as recited in any of Claims 11 to 13,
- 34 wherein said hydraulic means is configured to open each
- 35 check valve by the operation of said controller to
- 36 pressurize a selected hydraulic line.

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1 15. An apparatus as recited in any of Claims 11 to 14,

- 2 wherein said hydraulic means is configured to open a
- 3 selected combination of check valves by the operation
- 4 of said controller to pressurize a selected hydraulic
- 5 line.

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- 7 16. An apparatus as recited in any of Claims 11 to 15,
- 8 wherein said hydraulic means is configured to open each
- 9 check valve by the pressurization of one hydraulic
- 10 line.

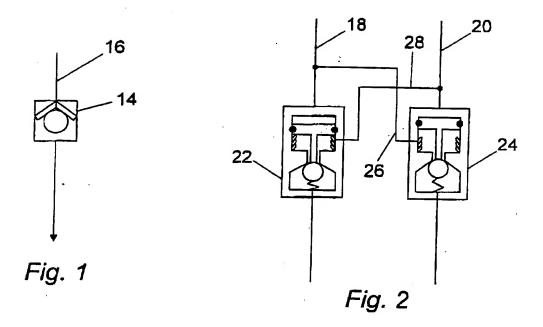
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- 12 17. An apparatus as recited in Claim 16, wherein said
- 13 hydraulic means is configured so that the
- 14 pressurization of each hydraulic line independently
- opens all of said check valves to two-way fluid
- 16 communication.

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- 18 18. An apparatus as recited in any of Claims 11 to 17,
- 19 wherein said controller is operable to withdraw
- 20 pressurization of said hydraulic lines to return each
- of said check valves to said closed initial position.

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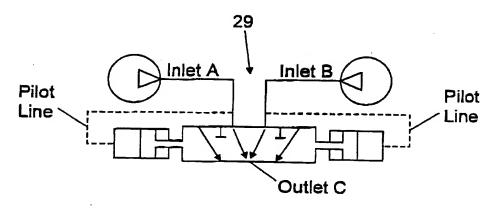


Fig. 3

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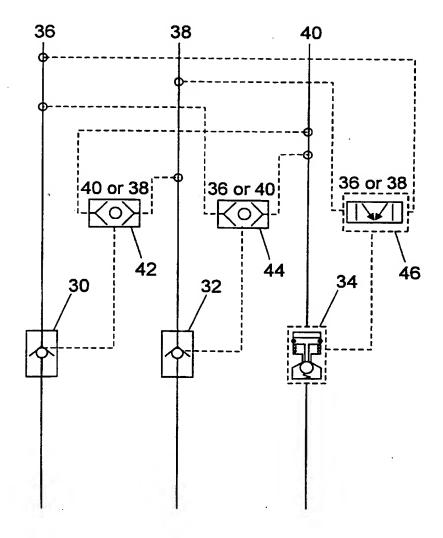


Fig. 4

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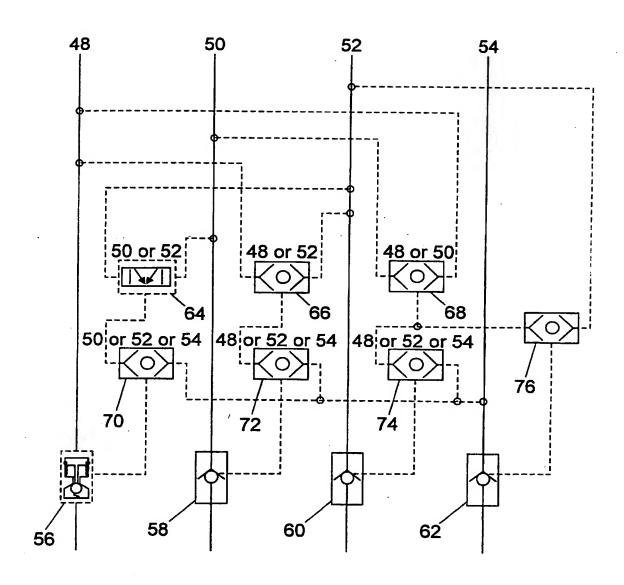


Fig. 5

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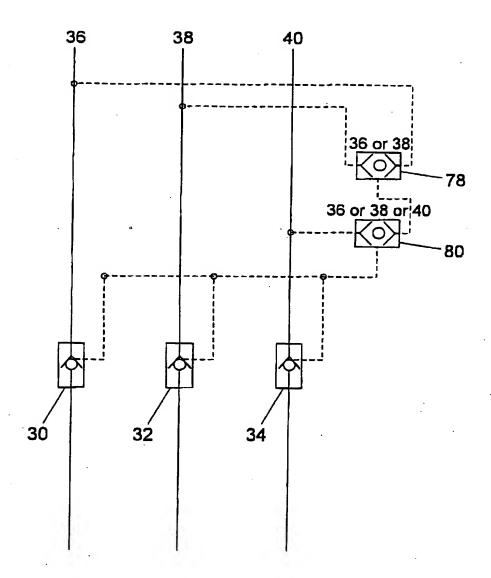


Fig. 6

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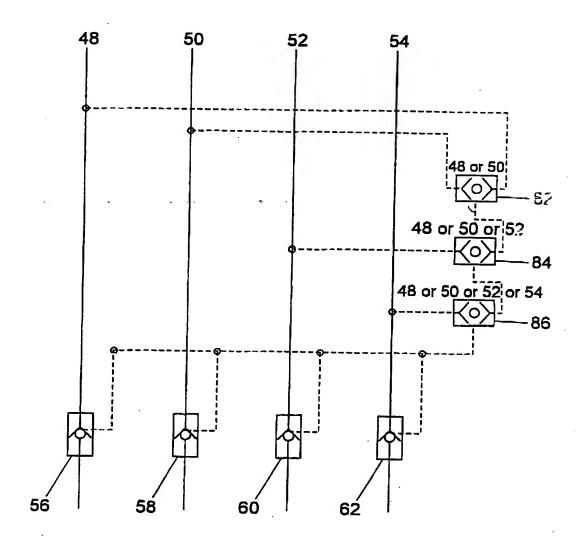


Fig. 7

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INTERNATIONAL SEARCH REPORT

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| | tion searched other than minimum documentation to the extent tha | | |
| Electronic | data base consulted during the international search (name of data i | base and, where practical, search terms used | |
| C. DOCUM | ENTS CONSIDERED TO BE RELEVANT | | |
| Category * | Citation of document, with indication, where appropriate, of the | relevant passages | Relevant to claim No. |
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